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# Numerical Solution of Laplace's Equation

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### Recommended Citation

Hansen, Per Brinch, "Numerical Solution of Laplace's Equation" (1992). *Electrical Engineering and Computer Science - Technical Reports*. 168.

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SU-CIS-92-17

***Numerical Solution of Laplace's Equation***

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September 1992

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# Numerical Solution of Laplace's Equation<sup>1</sup>

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September 1992

This tutorial discusses Laplace's equation for steady state heat flow in a two-dimensional region with fixed temperatures on the boundaries. The equilibrium temperatures are computed for a square grid using successive overrelaxation with parity ordering of the grid elements. The numerical method is illustrated by a Pascal algorithm. We assume that the reader is familiar with elementary calculus.

Categories and Subject Descriptors: G.1.8 [Numerical Analysis]: Partial Differential Equations—*difference methods*

General Terms: Algorithms

Additional Key Words and Phrases: Laplace's equation

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## INTRODUCTION

Physical phenomena that vary continuously in space and time are described by *partial differential equations*. The most important of these is *Laplace's equation*, which defines gravitational and electrostatic potentials as well as stationary flow of heat and ideal fluid [Feynman 1989].

This tutorial discusses Laplace's equation for steady state heat flow in a two-dimensional region with fixed temperatures on the boundaries. The equilibrium temperatures are computed on a square grid using *successive overrelaxation* with *parity ordering* of the grid elements [Young 1971, Press et al. 1989].

The numerical method is illustrated by a Pascal algorithm. A parallel version of this algorithm has been tested on a Computing Surface [Brinch Hansen 1992].

We assume that the reader is familiar with elementary calculus.

## 1. THE HEAT EQUATION

We will illustrate Laplace's equation by defining the equilibrium temperatures in a thin plate of homogeneous material and uniform thickness. The faces of the plate are perfectly insulated. On the boundaries of the plate, each point is kept at a known fixed temperature. As heat flows through the plate from warm towards cold boundaries, the plate eventually reaches a state where each point has a steady (time-independent) temperature maintained by the heat flow. The problem is to define the equilibrium temperature  $u(x, y)$  for every point  $(x, y)$  on the plate.

We will study the heat flow through a small rectangular element of the plate with origin  $(x, y)$  and sides of length  $\Delta x$  and  $\Delta y$  (Fig. 1).

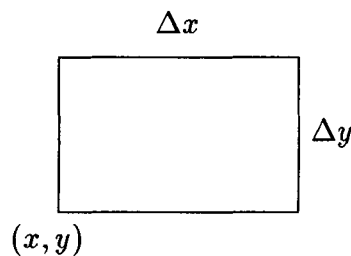


Fig. 1 A small element

Due to the insulation of the faces, the heat flows in two dimensions only. At every point  $(x, y)$  the velocity  $v$  of the heat flow has a horizontal component  $v_x$  and a vertical component  $v_y$  (Fig. 2). We will examine the horizontal and vertical flows separately.

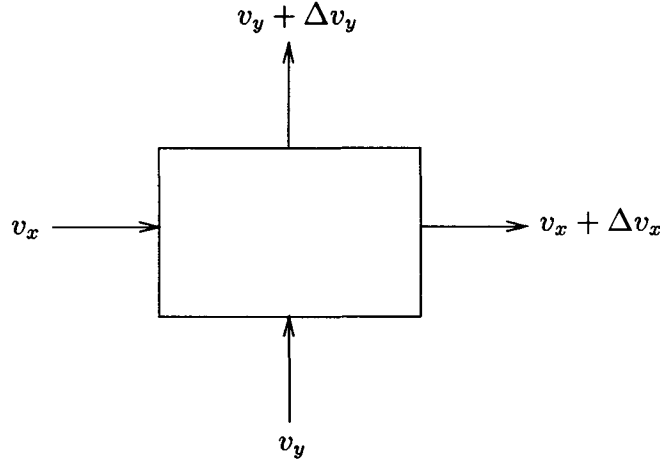


Fig. 2 Horizontal and vertical flow

The horizontal heat flow enters the element through the left boundary with velocity  $v_x$  and leaves the element through the right boundary with velocity  $v_x + \Delta v_x$ . Since these boundaries have length  $\Delta y$ , the element receives heat at the rate

$$v_x \Delta y$$

and loses heat at the rate

$$(v_x + \Delta v_x) \Delta y$$

So the horizontal flow removes heat from the element at the rate

$$\Delta v_x \Delta y = \frac{\partial v_x}{\partial x} \Delta x \Delta y$$

Similarly, the vertical flow removes heat from the element at the rate

$$\Delta v_y \Delta x = \frac{\partial v_y}{\partial y} \Delta y \Delta x$$

The combined rate of heat loss is the sum of the horizontal and vertical loss rates:

$$\left( \frac{\partial v_x}{\partial x} + \frac{\partial v_y}{\partial y} \right) \Delta x \Delta y$$

In equilibrium, the element holds a constant amount of heat, which keeps its temperature  $u(x, y)$  constant. Since the rate of heat loss is zero in the steady state, we have the *heat conservation* law:

$$\frac{\partial v_x}{\partial x} + \frac{\partial v_y}{\partial y} = 0 \quad (1)$$

Now, heat flows towards decreasing temperatures at a rate proportional to the temperature gradient:

$$v_x = -k \frac{\partial u}{\partial x} \quad v_y = -k \frac{\partial u}{\partial y} \quad (2)$$

where  $k$  is a constant [Feynman 1989]. This is the law of the *velocity potential*.

By combining the conservation and potential laws, we obtain *Laplace's equation* for equilibrium temperatures:

$$\frac{\partial^2 u}{\partial x^2} + \frac{\partial^2 u}{\partial y^2} = 0 \quad (3)$$

This partial differential equation is also known as the *heat equation* or the *equilibrium equation*. It is often written as follows:

$$\nabla^2 u = 0 \quad (4)$$

A function  $u(x, y)$  that satisfies this equation is called a *potential function*. The *boundary conditions* determine a potential function.

## 2. DIFFERENCE EQUATIONS

As an example, we will solve the heat equation for a *square region*, where the temperature is fixed at each boundary (Fig. 3).

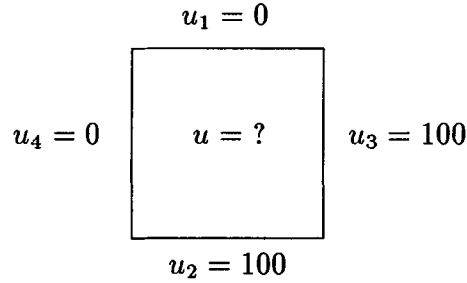


Fig. 3 A square region

A numerical solution can be found only for a *discrete* form of the heat equation. The first step is to divide the square region into a *square grid* of elements with uniform *spacing*  $h$  (Fig. 4). The center of each element represents a single point in the region. There are three kinds of grid elements:

1) *Interior elements*, marked “?”, have unknown temperatures, which satisfy a discrete heat equation.

2) *Boundary elements*, marked “+”, have fixed, known temperatures.

3) *Corner elements*, marked “—”, are not used.

−	+	+	+	+	−
+	?	?	?	?	+
+	?	?	?	?	+
+	?	?	?	?	+
+	?	?	?	?	+
−	+	+	+	+	−

Fig. 4 A square grid

A numerical solution of the heat equation is based on the observation that the heat flow through an interior element is driven by *temperature differences* between the element and its immediate neighbors. Figure 5 shows an interior element and four adjacent elements. The five elements are called *c* (center), *n* (north), *s* (south), *e* (east), and *w* (west).

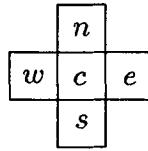


Fig. 5 Adjacent elements

These elements have the following *coordinates* and temperatures:

$$u_n = u(x, y + h)$$

$$u_w = u(x - h, y) \quad u_c = u(x, y) \quad u_e = u(x + h, y)$$

$$u_s = u(x, y - h)$$

If the grid spacing  $h$  is small enough, we can approximate the temperature  $u_e$  by the first three terms of a *Taylor expansion* of the function  $u$  in the neighborhood east of  $(x, y)$  [Courant and John 1989]:

$$u_e \approx u_c + h \frac{\partial u}{\partial x} + \frac{1}{2} h^2 \frac{\partial^2 u}{\partial x^2}$$

A Taylor expansion in the western neighborhood of  $(x, y)$  gives an approximation of the temperature  $u_w$ :

$$u_w \approx u_c - h \frac{\partial u}{\partial x} + \frac{1}{2} h^2 \frac{\partial^2 u}{\partial x^2}$$

Consequently,

$$u_e + u_w \approx 2u_c + h^2 \frac{\partial^2 u}{\partial x^2} \quad (5)$$

Similarly,

$$u_n + u_s \approx 2u_c + h^2 \frac{\partial^2 u}{\partial y^2} \quad (6)$$

From (5) and (6) we obtain the important approximation:

$$\nabla^2 u \approx (u_n + u_s + u_e + u_w - 4u_c)/h^2$$

According to the heat equation (4), the left-hand side is zero for steady state heat flow. Consequently, the discrete heat equation is a system of *difference equations* of the form:

$$4u_c - u_n - u_s - u_e - u_w \approx 0 \quad (7)$$

There is a separate equation for each of the  $n \times n$  internal elements.

In thermal *equilibrium*, the temperature of each grid element is simply the *average* of the temperatures of the four surrounding elements:

$$u_c \approx (u_n + u_s + u_e + u_w)/4 \quad (8)$$

### 3. NUMERICAL SOLUTION

On an  $n \times n$  grid, the heat equation is replaced by  $n^2$  linear equations in the unknown temperatures. Solving this system by a direct method requires  $O(n^6)$  run time on a sequential computer. For a grid of, say,  $250 \times 250$  elements, we obtain 62,500 linear equations with  $4 \times 10^9$  coefficients. If the run time is measured in units of 1  $\mu\text{sec}$ , the computation takes 8 years. This is clearly impractical.

Each equation (7) has five nonzero coefficients only. For such a *sparse* linear system, *iterative* methods are far more efficient than direct methods. Iterative solution of linear equations is known as *relaxation*. In the following, we develop a Pascal algorithm that uses relaxation to solve the discrete heat equation on a square grid with fixed boundary values.

A *grid*  $u$  is represented by an  $(n+2) \times (n+2)$  real array:

```
const n = 250;
type row = array [0..n+1] of real;
      grid = array [0..n+1] of row;
var u: grid;
```

It consists of  $n \times n$  interior elements surrounded by a border of  $4n$  boundary elements as shown in Fig. 4.

The *Laplace procedure* initializes the boundary elements with fixed temperatures  $u_1$ ,  $u_2$ ,  $u_3$ , and  $u_4$ , and the interior elements with the tentative temperature  $u_5$ . This is followed by a fixed number of relaxation steps (Algorithm 1).



```

procedure laplace(var u: grid; u1, u2,
    u3, u4, u5: real; steps: integer);
var k: integer;
begin
    newgrid(u);
    for k := 1 to steps do relax(u)
end

```

## Algorithm 1

A *new grid* holds known values in the boundary elements and reasonable values in the interior elements based on educated guessing (Algorithm 2).

```

procedure newgrid(var u: grid);
var i, j: integer;
begin
    for i := 0 to n + 1 do
        for j := 0 to n + 1 do
            u[i,j] := initial(i, j)
        end
    end

```

## Algorithm 2

The known and estimated temperatures shown in Fig. 3 are defined by the same *initial* function (Algorithm 3). The temperatures of the four corner elements are arbitrary (and irrelevant).

```

function initial(i, j: integer)
    : real;
begin
    if i = 0 then
        initial := u1
    else if i = n + 1 then
        initial := u2
    else if j = n + 1 then
        initial := u3
    else if j = 0 then
        initial := u4
    else
        initial := u5
    end

```

## Algorithm 3

The operation

```
laplace(u, 0.0, 100.0,
        100.0, 0.0, 50.0, n)
```

solves the heat flow problem shown in Fig. 3. As a reasonable guess, the interior temperatures are initially set at the average of the boundary temperatures:

$$(0 + 100 + 100 + 0)/4 = 50$$

#### 4. RELAXATION METHODS

A relaxation step replaces the temperature of every inner element by a better approximation based on the previous temperature of the element and the temperatures of its neighbors (see Fig. 5).

The simplest method is *Jacobi relaxation*, which conceptually updates every temperature simultaneously. This procedure must keep a copy of the current grid until the next temperatures have been computed (Algorithm 4).

```
procedure relax(var u: grid);
var u0: grid; i, j: integer;
begin
  u0 := u;
  for i := 1 to n do
    for j := 1 to n do
      u[i,j] := next(u0[i,j],
                    u0[i-1,j], u0[i+1,j],
                    u0[i,j+1], u0[i,j-1])
    end
  end
```

Algorithm 4

The most obvious idea is to replace every approximate temperature  $u_c$  by the average of the surrounding approximate temperatures  $u_n$ ,  $u_s$ ,  $u_e$ , and  $u_w$  (Algorithm 5).

```
function next(uc, un, us,
             ue, uw: real): real;
begin
  next := (un + us + ue +
           uw)/4.0
end
```

Algorithm 5

Algorithm 6 uses a new temperature as soon as it has been computed. This in-place computation cuts the memory requirement in half. The method is called

*Gauss-Seidel relaxation*, “even though Gauss didn’t know about it and Seidel didn’t recommend it” [Strang 1986].

```

procedure relax(var u: grid);
var i, j: integer;
begin
  for i := 1 to n do
    for j := 1 to n do
      u[i,j] := next(u[i,j],
        u[i-1,j], u[i+1,j],
        u[i,j+1], u[i,j-1])
    end
  end

```

#### Algorithm 6

Since new values are better approximations than old ones, Gauss-Seidel converges twice as fast as Jacobi. However, both methods require  $O(n^2)$  relaxation steps and  $O(n^4)$  run time [Press et al. 1989]. If the run time is measured in units of, say, 10  $\mu\text{sec}$ , the computation of  $250 \times 250$  equilibrium temperatures takes 11 hours. The slow convergence of these methods makes them impractical for scientific computing.

By 1950 engineers had discovered that convergence is much faster if temperatures are *overrelaxed*. Instead of approximating a new temperature by the average temperature

$$\text{aver} = (u_n + u_s + u_e + u_w)/4.0$$

we make the next temperature a weighted sum of the old temperature and the surrounding temperatures

$$\text{next} = (1 - f) * u_c + f * \text{aver}$$

The *relaxation factor*  $f$  is 1 for Gauss-Seidel. For overrelaxation,  $f > 1$ . This *ad hoc* method converges only if  $f < 2$  [Press et al. 1989].

It was a breakthrough when David Young [1954] developed a theory of overrelaxation. For the heat equation with fixed boundary values on a large  $n \times n$  square grid, the fastest convergence is obtained with the following relaxation factor:

$$f_{\text{opt}} = 2 - 2\pi/n$$

For  $n = 250$ , the optimal factor is  $f_{\text{opt}} = 1.97$ .

It is illuminating to look at overrelaxation from a different point of view. The *residual*

$$\text{res} = \text{aver} - u_c$$

is a measure of how well an approximate temperature  $u_c$  satisfies the discrete heat equation (7). In overrelaxation, the correction of each temperature is proportional to its residual:

$$\text{next} = u_c + f * \text{res}$$

where  $1 < f < 2$ .

The method of *successive overrelaxation* is based on this accidental discovery supported by Young's theory (Algorithm 7).

```

function next(uc, un, us,
  ue, uw: real): real;
var res: real;
begin
  res := (un + us + ue +
    + uw)/4.0 - uc;
  next := uc + fopt*res
end

```

Algorithm 7

The method requires  $n$  steps and  $O(n^3)$  run time to achieve 3-figure accuracy [Press et al. 1986]. If the run time is measured in units of  $10 \mu\text{sec}$ , a  $250 \times 250$  grid requires less than 3 minutes of computing.

The theory of optimal overrelaxation is valid only if the grid elements are updated in an appropriate order. The row-wise updating of Gauss-Seidel is acceptable. However, it is inherently a sequential method. Instead, we will use an ordering that can be adapted for parallel computing [Young 1971, Barlow and Evans 1982, Brinch Hansen 1992].

Every element  $u[i, j]$  has a *parity*

$$(i + j) \bmod 2$$

which is either *even* (0) or *odd* (1). Even and odd elements alternate on the grid like black and white squares on a chessboard. Every interior element is surrounded by four elements of the opposite parity (Fig. 6). So, the updating of the even elements depends only on the odd elements, and vice versa.

—	1	0	1	0	—
1	0	1	0	1	0
0	1	0	1	0	1
1	0	1	0	1	0
0	1	0	1	0	1
—	0	1	0	1	—

Fig. 6 Parity ordering

Algorithm 8 defines *successive overrelaxation* with *parity ordering*. A relaxation step consists of a scan of the even elements followed by a scan of the odd elements. A single scan updates all elements with the same *parity*  $b$ . We assume that  $n$  is *even*.

```

procedure relax(var u: grid);
var b, i, j, k: integer;
begin
  for b := 0 to 1 do
    for i := 1 to n do
      begin
        k := (i + b) mod 2;
        j := 2 - k;
        while j <= n - k do
          begin
            u[i,j] := next(u[i,j],
                          u[i-1,j], u[i+1,j],
                          u[i,j+1], u[i,j-1]);
            j := j + 2
          end
        end
      end
    end
  end

```

Algorithm 8

Successive overrelaxation can be refined further by *Chebyshev acceleration*, which changes the relaxation factor after each scan [Press et al. 1989]. However, the fastest relaxation method is the *multigrid* technique, which computes  $n^2$  temperatures recursively on finer and finer grids in  $O(n^2)$  time [Press and Teukolsky 1991].

## 5. PASCAL ALGORITHM

The complete Pascal algorithm shown below is composed of Algorithms 1–3 and 7–8.

```
const n = 250 {even};  
type row = array [0..n+1] of real;  
    grid = array [0..n+1] of row;  
  
procedure laplace(var u: grid; u1, u2,  
    u3, u4, u5: real; steps: integer);  
const pi = 3.14159265358979;  
var fopt: real; k: integer;
```

```
    function initial(i, j: integer)  
        : real;  
    begin  
        if i = 0 then  
            initial := u1  
        else if i = n + 1 then  
            initial := u2  
        else if j = n + 1 then  
            initial := u3  
        else if j = 0 then  
            initial := u4  
        else  
            initial := u5  
    end;
```

```
    function next(uc, un, us,  
        ue, uw: real): real;  
    var res: real;  
    begin  
        res := (un + us + ue +  
            + uw)/4.0 - uc;  
        next := uc + fopt*res  
    end;
```

```
procedure newgrid(var u: grid);  
var i, j: integer;  
begin  
    for i := 0 to n + 1 do  
        for j := 0 to n + 1 do  
            u[i,j] := initial(i, j)  
end;
```

```

procedure relax(var u: grid);
var b, i, j, k: integer;
begin
  for b := 0 to 1 do
    for i := 1 to n do
      begin
        k := (i + b) mod 2;
        j := 2 - k;
        while j <= n - k do
          begin
            u[i,j] := next(u[i,j],
                          u[i-1,j], u[i+1,j],
                          u[i,j+1], u[i,j-1]);
            j := j + 2
          end
        end
      end
    end;

begin
  fopt := 2.0 - 2.0*pi/n;
  newgrid(u);
  for k := 1 to steps do relax(u)
end

```

## SUMMARY

We have derived Laplace's equation for steady-state heat flow in two dimensions and have explained how the equation is solved by successive overrelaxation on a discrete grid. The numerical method was illustrated by a Pascal algorithm.

## ACKNOWLEDGEMENTS

I thank Jonathan Greenfield for his comments.

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